

**In the Specification:**

Please amend the paragraph beginning at page 10, line 6 as follows:

--Various other features and attendant advantages of the present invention will be more fully appreciated as the same becomes better understood when considered in conjunction with the accompanying drawings, in which like reference characters designate the same or similar parts throughout the several views, and wherein:

Figure 1 is a process diagram of a low-temperature air-separation system;

Figures 2 to 4 show the arrangement of the distribution passages in conventional plate heat exchangers;

Figure 5 shows the arrangement of the heat exchange passages for a plate heat exchanger according to the invention;

Figure 6 shows a variant of the embodiment according to Figure 5;

Figures 7 and 8 show the division of a heat exchanger according to the invention into two component areas;

Figure 9 is a process diagram of an air-separation system with a single-turbine air cycle;

Figure 10 shows a process diagram of an air-separation system with a dual-turbine air cycle;

Figure 11 shows the arrangement of the heat exchange passages of the main heat exchanger according to the invention in the process according to Figure 9; and

Figure 12 shows the arrangement of the heat exchange passages of the main heat exchanger according to the invention in the process as shown in Figure 10;

Figure 13 is an isometric view of the heat exchanger illustrated in Figure 5; and

Figure 14 is a cross-sectional view of a heat exchanger according to the invention;

Figure 15 illustrates an isometric view, similar to Figure 13, wherein each of the component areas extends over only part of the width of the heat exchanger core and over only part of the depth of the heat exchanger core.--

Please amend the paragraph beginning at page 17, lines 10 as follows:

-- Figure 13 illustrates an isometric view of the heat exchanger of Figure 5. As Ass shown on the right side of Figure 13, the heat exchanger core comprises a plurality of parallel separating plates 111. As shown in Figure 14, spaces between adjacent plates (110 and 112) define passageways for fluid flow and heat transfer. As discussed previously, these passageways 110 and 112 can be divided into groups of heat exchange passages by fins positioned between the separating plates 111. Fluid flows 10, 20, 30, 40, 50, and 60 enter the heat exchanger via distributors 11, 21, 31, 41, and 51, respectively. The fluids 10, 20, 30, 40, 50, and 60 flow through the heat exchanger core 9 via the passageways defined between plates 111 and then are removed from the heat exchanger via collectors 12, 22, 32, 42, 52, and 62, respectively.--

At page 18, between lines 13 and 14, please insert the following paragraph:

--Figure 15 illustrates an isometric view of a heat exchanger similar to that of Figure 5, but in which the heat exchanger is divided along its depth as well as its width so that flows 30, 40 and 50 can each be two separate flows. Thus, fluid flows enter the heat exchanger via distributors 11, 21, 31A, 31B, 41A, 41B, 51A, 51B, and 61. The fluids flow through the heat exchanger core 9 via the passageways defined between plates 111 (see Figure 14) and then are removed from the heat exchanger via collectors 12, 22, 32A, 32B, 42A, 42B, 52A, 52B, and 62, respectively.--